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**Localized Bending Waves in an Elastic Orthotropic Plate**  
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**Локализованные изгибные волны в**  
**упругой ортотропной пластинке**

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В работе приведены результаты исследования вопроса существования изгибных локализованных волн в окрестности свободного края прямоугольной ортотропной пластинки.

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Լոկալիզացված ծոծան ալիքների առաձգական օրթոտրոպ սալում

Աշխատանքում բերված են ուղղանկյուն օրթոտրոպ սալի ազատ եզրի շրջակայքում ծոծան լոկալիզացված ալիքների գոյության խնդրի արդյունքները:

The aim of this work is the theoretical study of bending localized waves in a thin elastic orthotropic cantilever plate. These waves are spatially non-uniform bending perturbations varying in time, localized near the vicinity of limiting free surface of a plate and practically immaterial outside this relatively narrow zone. The first studies related to the localized bending waves in elastic plates were first presented in [1] and further developed in [2], where for semi-infinite plate the existence of surface bending waves near the free edge have been shown. For two semi-infinite plates being in conditions of elastic contact, a similar problem was investigated in [3].

The dynamic problem is considered for elastic bending propagation waves in an orthotropic cantilever plate, one edge of which is free from mechanical stresses and restrictions.

In Cartesian system  $(x, y)$ , where the plate occupy domain  $x \in [0, a]$ ,  $y \in [0, b]$  the equation for plate middle plane normal displacement  $W(x, y)$  can be expressed as [4]

$$\frac{\partial^4 W}{\partial \alpha^4} + 2k \frac{\partial^4 W}{\partial \alpha^2 \partial \beta^2} + \frac{\partial^4 W}{\partial \beta^4} + \frac{3\rho(1 - \nu_1 \nu_2)}{h^2} \frac{\partial^2 W}{\partial t^2} = 0 \quad (1)$$

In (1)  $\alpha = xE_1^{-1/4}$ ,  $\beta = yE_2^{-1/4}$ ,  $k = (E_1 E_2)^{-1/2} [\nu_1 E_1 + 2(1 - \nu_1 \nu_2)G]$ ,  $E_1, E_2, G, \nu_1, \nu_2$  are elastic constants,  $\rho$  is the bulk density of plate material,  $h$  and is the thickness of plate. In the case of an isotropic plate one has  $k = 1$ . For cantilever plate the equation (1) is supplemented by the following boundary conditions:

Free edge (the bending moment and the generalized transverse force are vanished)

$$\frac{\partial^2 W}{\partial \beta^2} + \nu \frac{\partial^2 W}{\partial \alpha^2} = 0; \quad \frac{\partial}{\partial \beta} \left[ \frac{\partial^2 W}{\partial \beta^2} + (2k - \nu) \frac{\partial^2 W}{\partial \alpha^2} \right] = 0 \text{ at } \beta = 0 \quad (2)$$

Where the following notation are used

$$\nu = \frac{\nu_1 E_1}{\sqrt{E_1 E_2}} = \frac{\nu_2 E_2}{\sqrt{E_1 E_2}}$$

Simply supported edges (the displacement and bending moment are vanished)

$$W = 0; \quad \frac{\partial^2 W}{\partial \beta^2} = 0, \quad \text{at } \beta = bE_2^{-1/4} \quad (3)$$

$$W = 0; \quad \frac{\partial^2 W}{\partial \alpha^2} = 0, \quad \text{at } \alpha = 0; \quad \alpha = aE_1^{-1/4} \quad (4)$$

In (1-4) a time-harmonic plane wave solution be considered

$$W(\alpha, \beta, t) = W_0(\beta) \sin(p_n \alpha) \exp(i\omega t) \quad (5)$$

where  $\omega$  is the vibration frequency,  $p_n = \pi n E_1^{1/4} / a$

Substituting (5) in (2-4) we have the following self-adjoint eigenvalue boundary problem for displacement function  $W_0(\beta)$

$$\frac{1}{p_n^4} \frac{d^4 W}{d\beta^4} - \frac{2k}{p_n^2} \frac{d^2 W}{d\beta^2} = \lambda W \quad \beta \in [0, bE_2^{-1/4}]$$

$$\frac{d^2 W}{d\beta^2} - \nu p_n^2 W = 0; \quad \frac{d}{d\beta} \left[ \frac{d^2 W}{d\beta^2} - (2k - \nu) p_n^2 W \right] = 0 \text{ at } \beta = 0 \quad (6)$$

$$W = 0; \quad \frac{d^2 W}{d\beta^2} = 0 \text{ at } \beta = bE_2^{-1/4}$$

Here  $\lambda = \frac{3\rho(1 - \nu_1 \nu_2) \omega^2}{h^2 p_n^4} - 1$  are eigenvalues of the boundary problem (6).

Negative eigenvalues  $\lambda$  of the boundary problem, if they exist, define the localized mode of vibration [5]; positive eigenvalues correspond to periodic modes.

Using common procedure of self adjoint boundary value problem solution [2,6,7] we have the following equation determining the frequencies of localized mode of vibration for negative eigenvalues  $\mu = -|\lambda|$

$$F^{(+)}(\mu) = F^{(-)}(\mu) \quad (7)$$

where

$$F^{(\pm)}(\mu) = \frac{(k - \nu \pm \sqrt{k^2 - \mu})^2}{\sqrt{k \pm \sqrt{k^2 - \mu}}} \operatorname{th} \left( \gamma_n \sqrt{k \pm \sqrt{k^2 - \mu}} \right); \gamma_n = (E_1/E_2)^{1/4} b \pi n / a$$

In the case of elongated plate  $b/a \gg 1$  replacing function

$$\operatorname{th} \left( \gamma_n \sqrt{k \pm \sqrt{k^2 - \mu}} \right) \rightarrow 1$$

we can obtain the results of [6].

Based on equation (7) the necessary and sufficient conditions are obtained regarding localized wave existence in depend of on anisotropy coefficients  $k, \nu$ . It is shown that equation (7) may have only one root that correspond to localised bending wave.

#### References

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